A New Approach for Motion Control of Constrained Mechanical Systems: Using Udwadia-Kalaba's Equations of Motion

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ABSTRACT

A new approach for motion control of constrained mechanical systems is proposed in this paper. The approach uses a new equations of motion which is proposed by Udwadia and Kalaba and named Udwadia-Kalaba's equations of motion in this paper. This paper reveals that the Udwadia-Kalaba's equations of motion is more adequate to model constrained mechanical systems rather than the famous Lagrange's equations of motion at least for control purpose. The proposed approach covers most of constraints including holonomic and nonholonomic constraints. Comparison of simulation results of two systems which are well-known in the literature show the superiority of the proposed approach. Furthermore, a special constrained mechanical system which includes nonlinear generalized velocities in its constraint equations, which has been considered to be difficult to control, can be controlled easily. It shows the possibility of the proposed approach to being a general framework for motion control of constrained mechanical systems with various kinds of constraints.

Keywords: Udwadia-Kalaba's equations of motion, constrained mechanical systems, holonomic and nonholonomic constraint, Moore-Penrose generalized inverse, nonlinear generalized velocity, generalized constraint force

1. Introduction

Interests on the control of mechanical systems with kinematic constraints are increasing recently. The constraints in the category encompass usually holonomic and nonholonomic constraints. Numerous papers have been published on the control of mechanical systems with holonomic constraints (1)(2)(3). On the other hand, control of mechanical systems with nonholonomic constraints is investigated somewhat recently and relatively small number of papers are found in literature (4)(5)(6). Furthermore, they usually deals with some specific examples.

Traditionally, Lagrangian mechanics is adapted for the modeling of mechanical systems with holonomic or nonholonomic constraints. It requires the use of Lagrangian multipliers which in turn means force measurement when one needs to feedback the

Lagrangian multipliers. It becomes usually a source of uncertainty and is desirable to avoid it in the case of motion control.

Udwadia and Kalaba⁽⁷⁾ proposed a new method which deals general equations of motion for constrained discrete dynamic systems. Explicit equations of motion and generalized force of constraint can be obtained for the system with constraints of the form $A(q,\dot{q},t)\ddot{q}=b(q,\dot{q},t)$ where A is a known $m\times n$ matrix, b is a known m vector, and q is n generalized coordinates. It includes, among others, the usual holonomic and nonholonomic constraints.

In this paper, the Udwadia and Kalaba's equation is modified for the motion control design of constrained mechanical systems. In Section II, the Udwadia and Kalaba's equation is summarized briefly. The equation if modified in Section III for the motion control design of constrained mechanical systems. New dynamics of the constrained mechanical system and new input matrix are

introduced. Two examples from literature are simulated in Section IV for verification of the use of Udwadia and Kalaba's equation for mechanical systems with nonholonomic constraints. Section V concludes this paper with concluding remarks.

2. Background Material

Udwadia and Kalaba⁽⁷⁾ proposed a novel method which yields the explicit general equations of motion for constrained discrete dynamic systems. The method can handle many kinds of constraints including holonomic and nonholonomic constraints. Main results of the method are briefly reviewed in this section for later use.

The equation of motion of an unconstrained discrete dynamic system can be represented as

$$M(q,t)\ddot{q} = Q(q,\dot{q},t) \tag{1}$$

where $q \in \mathbb{R}^n$ is the generalized coordinates, $\dot{q} = \frac{d}{dt}q$, $\ddot{q} = \frac{d}{dt}\dot{q}$, the $n \times n$ matrix M is symmetric and positive definite, and n-vector Q is the portion of the system that is not directly related to the acceleration \ddot{q} .

Suppose that the system is subjected to the following constraints,

$$A(q,\dot{q},t)\ddot{q} = b(q,\dot{q},t) \tag{2}$$

where A is a known $m \times n$ constraint matrix and b is a known m -vector. It can be seen that (2) includes holonomic, nonholonomic, and many other kinds of constraints.

The explicit equations of motion of a discrete dynamic system (1) under (2) can be written as

$$M(q,t)\ddot{q} = Q(q,\dot{q},t) + Q_c(q,\dot{q},t)$$
(3)

where n-vector $Q_c(q,\dot{q},t)$ represents the generalized constraint forces.

The first main result of Udwadia and Kalaba⁽⁷⁾ is the derivation of the explicit equations of motion under constraints (2) as

$$M\ddot{q} = Q + K(b - AM^{-1}Q) \tag{4}$$

where the $m \times n$ matrix $K(q, \dot{q}, t) = M^{1/2} (AM^{-1/2})^+$

and where is $M^{1/2}$ a unique positive definite square root of a positive definite matrix M and '+' denotes the Moore-Penrose generalized inverse (see Penrose⁽⁸⁾).

The second main result is the derivation of the explicit equations for the generalized constraint forces as

$$Q_c(q,\dot{q},t) = K(b - AM^{-1}Q).$$
 (5)

It is worth to note that equations (4) and (5) are derived independent to any specific problem.

3. A New Model for Motion Control of Constrained Mechanical Systems

In this section, a new modeling method for motion control of constrained mechanical systems is derived using the Udwadia-Kalaba approach. The constraints can be holonomic, nonholonomic, and many other forms even nonlinear in q, \dot{q} , and t.

Consider a mechanical system to be controlled whose model is represented as

$$M(q,t)\ddot{q} + G(q,\dot{q},t) = \tau \tag{6}$$

where n -vector τ is the external forcing term. Suppose that the system is under the constraint

$$f(q,\dot{q},t) = 0 \tag{7}$$

where $f(\cdot): \mathbb{R}^n \times \mathbb{R}^n \times \mathbb{R} \to \mathbb{R}^m$ is assumed to be \mathbb{C}^1 (i.e., differentiable).

The control problem is to find, which may be dependent on q, \dot{q} , and t, such that one may achieve a given task while observing the constraint for all $t \ge t_0$, t_0 is the initial time. The constrained mechanical system (6) and (7) can be rewritten like the form of (1) and (2) in order to use the Udwadia-Kalaba approach. First, the unconstrained equations of motion (6) can be rewritten as

$$M(q,t)\ddot{q} = \tau - G(q,\dot{q},t). \tag{8}$$

Comparison of (1) and (8) gives

$$Q(q,\dot{q},t) = \tau - G(q,\dot{q},t). \tag{9}$$

Secondly, the constraint (7) can be represented like the form of (2) by differentiating it with respect to t .

Differentiation of $f(q, \dot{q}, t)$ in (7) with respect to t yields

$$\frac{df}{dt} = \frac{\partial f}{\partial q} \frac{dq}{dt} + \frac{\partial f}{\partial \dot{q}} \frac{d\dot{q}}{dt} + \frac{\partial f}{\partial t}$$
 (10)

or

$$\frac{df}{dt} = \frac{\partial f}{\partial q}\dot{q} + \frac{\partial f}{\partial \dot{q}}\ddot{q} + \frac{\partial f}{\partial t}$$
 (11)

Therefore equations (2), (7), and (11) give

$$A(q,\dot{q},t) = \frac{\partial f}{\partial \dot{q}}$$
 and $b(q,\dot{q},t) = \frac{\partial f}{\partial q}\dot{q} - \frac{\partial f}{\partial t}$. (12)

Note that the derivatives such that $\frac{\partial f}{\partial q}$ are to be interpreted as vector forms since the dimensions of f,q and \dot{q} can be more than one. So, a mechanical system in (6) which is to be controlled under the constraints (7) is represented like equations (1) and (2) in the Udwadia-Kalaba approach through (8) and (12).

Application of (4) to the mechanical system with (8) and (12) yields

$$M\ddot{q} = \tau - G + M^{1/2}C^{+}[b - AM^{-1}(\tau - G)]$$
 (13)

where $C = AM^{-1/2}$ of course, the existence of M^{-1} is assumed in the approach.

Equation (13) can be rearranged as

$$M\ddot{q} + G - M^{1/2}C^{+}(b + AM^{-1}G)$$

$$= (1 - M^{1/2}C^{+}AM^{-1})\tau.$$
(14)

The left-hand-side of (14) is the "new" dynamics of the constrained mechanical system and the right-hand-side is the control with "new" input matrix. Usually, the rank of $I - M^{1/2}C^+AM^{-1}$ is lower than that of I and the system is "underactuated". It means that the system can be feedback stabilized to an equilibrium manifold with smooth feedback in this case. It is pointed out in Bloch and McClamroch⁽⁴⁾, Campion *et al.*⁽⁵⁾, Bloch *et al.*⁽⁶⁾, and Su and Stepanenko⁽⁹⁾.

It is worth to note that (14) is decoupled with the generalized constraint forces. It means that the motion of a mechanical system under constraints can be controlled using only position and velocity feedback, i.e., it does

not need force sensing.

Application of (5) to the mechanical system with (8) and (12) gives

$$Q_c = M^{1/2}C^+(b + AM^{-1}G) - M^{1/2}C^+AM^{-1}\tau$$
. (15)

Equation (15) gives the possibility of the control of the generalized constraint forces if desired.

The new modeling method is suitable for motion control since the constraints are embedded into the "new" dynamic equation. The validity of using the Udwadia-Kalaba approach for motion control purpose is demonstrated in the following section with some simulated examples.

4. Simulated Examples

Validity and efficiency of using the Udwadia-Kalaba approach to the control of mechanical system with non-holonomic constraint is demonstrated in the example 1.

Example 1. Equations of motion of a wheeled robot moving on a horizontal plane in Su and Stepanenko⁽⁹⁾ with L = P = 1 are adopted in this example, where P is the radius of the wheels and 2L is the length of the axis of the front wheels. It is constituted by a rigid trolley equipped with non-deformable wheels. The physical configuration and derivation of the equations can be found in Campion *et al.*⁽⁵⁾ and d'Andrea-Novel *et al.*⁽¹⁰⁾.

The unconstrained equations of motion can be expressed as

$$M\ddot{x} = -(u_1 + u_2)\sin\theta$$

$$m\ddot{y} = (u_1 + u_2)\cos\theta$$

$$I_0\ddot{\theta} = u_1 - u_2$$
(16)

where x and y are generalized coordinates which represent the position of a reference point of the robot. θ is a generalized coordinate for orientation of the robot. m is the mass of the robot, I_0 is its inertia with respect to a vertical axis which is passing through the reference point, and u_1 and u_2 are control inputs.

Comparison of (16) with (8) and (9) yields and

$$M = \begin{bmatrix} m & 0 & 0 \\ 0 & m & 0 \\ 0 & 0 & I_0 \end{bmatrix} \text{ and } Q = \begin{bmatrix} -(u_1 + u_2)\sin\theta \\ (u_1 + u_2)\cos\theta \\ u_1 - u_2 \end{bmatrix}$$
 (17)

The nonholonomic constraint is expressed as

$$\dot{x}\cos\theta + \dot{y}\sin\theta = 0. \tag{18}$$

Differentiation of (18) with respect to t gives

$$\begin{bmatrix} \cos \theta & \sin \theta & 0 \end{bmatrix} \ddot{\ddot{y}} = \dot{x}\dot{\theta}\sin \theta - \dot{y}\dot{\theta}\cos \theta . \tag{19}$$

Comparison of (19) with (2) yields

$$A = \begin{bmatrix} \cos \theta & \sin \theta & 0 \end{bmatrix} \text{ and }$$

$$b = \dot{x}\dot{\theta}\sin \theta - \dot{y}\dot{\theta}\cos \theta.$$
(20)

Substitution of (17) and (20) into (4) gives

$$\begin{bmatrix} m & 0 & 0 \\ 0 & m & 0 \\ 0 & 0 & I_0 \end{bmatrix} \begin{bmatrix} \ddot{x} \\ \ddot{y} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} -\sin\theta & -\sin\theta \\ \cos\theta & \cos\theta \\ 1 & -1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}$$

$$+ m \begin{bmatrix} \cos\theta \\ \sin\theta \\ 0 \end{bmatrix} (\dot{x}\dot{\theta}\sin\theta - \dot{y}\dot{\theta}\cos\theta).$$
(21)

Equation (21) represents a new equations of motion where the nonholonomic constraint (18) is embedded. When the outputs are chosen as $[y \ \theta]^T$ the governing equations of motion become

$$\begin{bmatrix} m & 0 \\ 0 & I_0 \end{bmatrix} \begin{bmatrix} \ddot{y} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} \cos \theta & \cos \theta \\ 1 & -1 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} + m \begin{bmatrix} \sin \theta \\ 0 \end{bmatrix} (\dot{x}\dot{\theta}\sin \theta - \dot{y}\dot{\theta}\cos \theta).$$
 (22)

If the control task is to drive y, \dot{y}, θ , and $\dot{\theta}$ to $y = \dot{y} = \theta = \dot{\theta} = 0$, can choose u_1 and u_2 as

$$\begin{bmatrix} u_1 \\ u_2 \end{bmatrix} = \begin{bmatrix} \cos \theta & \cos \theta \\ 1 & -1 \end{bmatrix}^{-1} \left(-m \begin{bmatrix} \sin \theta \\ 0 \end{bmatrix} (\dot{x}\dot{\theta}\sin \theta - \dot{y}\dot{\theta}\cos \theta) + \begin{bmatrix} -l_1 y - l_2 \dot{y} \\ -m_1 \theta - m_2 \dot{\theta} \end{bmatrix} \right), \theta \neq \pm \frac{\pi}{2}, \pm \frac{3\pi}{2}, \dots$$
 (23)

where $l_1, l_2, m_{1,}$ and m_2 can be determined to satisfy desired performances. This results in

$$\begin{bmatrix} m & 0 \\ 0 & I_0 \end{bmatrix} \begin{bmatrix} \ddot{y} \\ \ddot{\theta} \end{bmatrix} = \begin{bmatrix} -l_1 y - l_2 \dot{y} \\ -m_1 \theta - m_2 \dot{\theta} \end{bmatrix}. \tag{24}$$

Equation (24) assures that the control task can be achieved.

A simulation is performed to the system (21) using the control (23). System parameters and initial conditions are the same as in Su and Stepanenko⁽⁹⁾ for comparison, i.e., m = 0.5, $I_0 = 0.5$, x(0) = 0, $\dot{x}(0) = 0$, y(0) = 4, $\dot{y}(0) = 0$, $\theta(0) = 45^{\circ}$ and $\dot{\theta}(0) = 0$. Control parameters

Figure 1 and 2 show that the control task is achieved satisfactorily. Much smoother and smaller velocity (in magnitude) than Su and Stepanenko⁽⁹⁾ are obtained even though the position and orientation trajectories are similar. Figure 3 shows the control profiles. Much smoother and smaller controls can be observed also. Figure 4 shows that the nonholonomic constraint is satisfied during the motion control.

Another simulated example is investigated for a

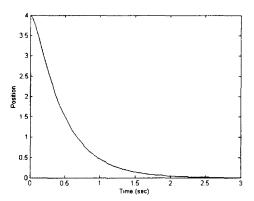


Fig. 1(a) Position trajectory of y

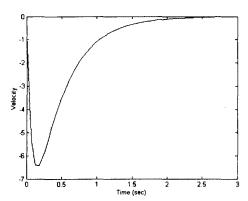


Fig. 1(b) Velocity profile of \dot{y}

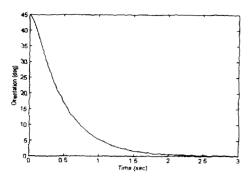


Fig. 2(a) Orientation trajectory of θ

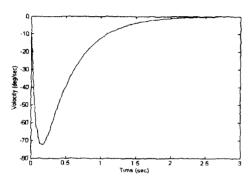


Fig. 2(b) Velocity profile of $\dot{\theta}$

system with a constraint which is nonlinear in generalized velocities. So far no other modeling technique is able to address such constraint.

Example 2. Consider a simple planar Cartesian manipulator in McClamroch and Wang⁽¹⁾. The unconstrained equations of motion are expressed as

$$\begin{bmatrix} \ddot{q}_1 \\ \ddot{q}_2 \end{bmatrix} = \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} \tag{25}$$

where q_1 and q_2 are generalized coordinates and u_1 and u_2 are control inputs.

Udwadia-Kalaba's equations of motion Comparison of (25) with (8) and (9) yields

$$M = \begin{bmatrix} 1 & 0 \\ 0 & 1 \end{bmatrix} \text{ and } Q = \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}. \tag{26}$$

The constraint equation is given by

$$4q_1^2 + q_2^2 - 1 = 0. (27)$$

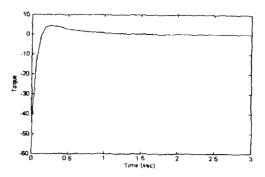


Fig. 3(a) Control profile of u_1

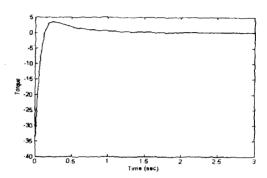
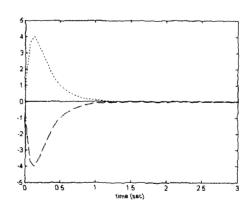


Fig. 3(b) Control profile of u_2



 $(\cdots \dot{x}\cos\theta, --: \dot{y}\sin\theta, --: \dot{x}\cos\theta + \dot{y}\sin\theta)$

Fig. 4. Evaluation of constraint $\dot{x}\cos\theta + \dot{y}\sin\theta$

The constraint, after taking derivative with respect to t twice, is

$$\left[8q_1 \quad q_2 \begin{bmatrix} \ddot{q}_1 \\ \ddot{q}_2 \end{bmatrix} = -8\dot{q}_1^2 - \dot{q}_2^2 \,.$$
 (28)

Comparison of (28) with (2) yields

$$A = \begin{bmatrix} 8q_1 & q_2 \end{bmatrix}$$
 and $b = -8\dot{q}_1^2 - \dot{q}_2^2$. (29)

Substitution of (26) and (29) into (4) gives

$$\begin{bmatrix} \ddot{q}_1 \\ \ddot{q}_2 \end{bmatrix} = \frac{1}{64q_1^2 + q_2^2} \begin{bmatrix} q_2^2 & -8q_1q_2 \\ -8q_1q_2 & 64q_1^2 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix} + \frac{-(8\dot{q}_1^2 + q_2^2)}{64q_1^2 + q_2^2} \begin{bmatrix} 8q_1 \\ q_2 \end{bmatrix}$$
(30)

Apparently the numerator $64q_1^2+q_2^2$ is nonzero since $4q_1^2+q_2^2-1=0$ or $64q_1^2+16q_2^2-16=0$. Equation (30) represents a new equations of motion where the constraint (27) is embedded. Note that the determinant of the "new" input matrix is zero, i.e., rank=1. Therefore, the two control u_1 and u_2 are dependent. We propose to choose

$$u_1 = u_2 = u. (31)$$

The system is

$$\begin{bmatrix} \ddot{q}_1 \\ \ddot{q}_2 \end{bmatrix} = \frac{1}{64q_1^2 + q_2^2} \begin{bmatrix} q_2^2 & -8q_1q_2 \\ -8q_1q_2 & 64q_1^2 \end{bmatrix} u + \frac{-(8\dot{q}_1^2 + \dot{q}_2^2)}{64q_1^2 + q_2^2} \begin{bmatrix} 8q_1 \\ q_2 \end{bmatrix}.$$
(32)

Let the output variables, i.e., the manipulated variable, be q_1 and \dot{q}_2 . The dynamics governing the output are

$$\begin{bmatrix} \dot{q}_1 \\ \ddot{q}_1 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ 0 & 0 \end{bmatrix} \begin{bmatrix} q_1 \\ \dot{q}_2 \end{bmatrix} + \frac{1}{64q_1^2 + q_2^2} \begin{bmatrix} 0 \\ q_2^2 - 8q_1q_2 \end{bmatrix} u + \begin{bmatrix} 0 \\ 8q_1 \end{bmatrix} \frac{-(8\dot{q}_1^2 + \dot{q}_2^2)}{64q_1^2 + q_2^2}.$$
(33)

Choose the control u to be the follow:

$$u = \frac{64q_1^2 + q_2^2}{q_2^2 - 8q_1q_2} \left(-k_1q_1 - k_2\dot{q}_1 + \frac{8\dot{q}_1^2 + \dot{q}_2^2}{8q_1(64q_1^2 + q_2^2)} \right). \tag{34}$$

The resulting closed-loop system is

$$\begin{bmatrix} \dot{q}_1 \\ \ddot{q}_1 \end{bmatrix} = \begin{bmatrix} 0 & 1 \\ -k_1 & -k_2 \end{bmatrix} \begin{bmatrix} q_1 \\ \dot{q}_1 \end{bmatrix}. \tag{35}$$

The system can be made asymptotically stable by any choice of $k_1 > 0$ and $k_2 > 0$. This renders $q_1 \to 0$ and $q_2 \to 0$ as $t \to \infty$.

If the task is to drive and then the control law needs to be modified:

$$u = \frac{64q_1^2 + q_2^2}{q_2^2 - 8q_1q_2} \left(-k_1q_1 - k_2\dot{q}_1 + (8\dot{q}_1^2 + \dot{q}_2^2) / (8q_1(64q_1^2 + q_2^2)) + k_1q_1d + k_2\dot{q}_1d \right)$$
(36)

It is assumed that q_{1d} and \dot{q}_{1d} are constants. It shows that the problem in McClamroch and Wang⁽¹⁾ can be handled easily.

Now, suppose that the constraint is changed to

$$\frac{1}{2}(\dot{q}_1^2 + \dot{q}_2^2) = 1\tag{37}$$

This implies that the kinetic energy of the system, even under the external control u_1 and u_2 , are kept constant. Note that the constraint is *nonlinear* in \dot{q}_1 and \dot{q}_2 .

Differentiation of the constraint with respect to t yields

$$\frac{1}{2}(2\dot{q}_1\ddot{q}_1 + 2\dot{q}_2\ddot{q}_2) = 0 \tag{38}$$

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$$\left[\dot{q}_1 \quad \dot{q}_2 \right] \begin{bmatrix} \ddot{q}_1 \\ \ddot{q}_2 \end{bmatrix} = 0.$$
 (39)

Comparison of (39) with (2) yields

$$A = \begin{bmatrix} \dot{q}_1 & \dot{q}_2 \end{bmatrix} \text{ and } b = 0. \tag{40}$$

Substitution of (26) and (40) into (4) gives

$$\begin{bmatrix} \ddot{q}_1 \\ \ddot{q}_2 \end{bmatrix} = \frac{1}{2} \begin{bmatrix} q_2^2 & -\dot{q}_1\dot{q}_2 \\ -\dot{q}_1\dot{q}_2 & \dot{q}_1^2 \end{bmatrix} \begin{bmatrix} u_1 \\ u_2 \end{bmatrix}. \tag{41}$$

Equation (41) represents a new equations of motion where the nonlinear constraint (37) in embedded. The two control u_1 and u_2 are dependent since the input matrix is of rank 1. We propose to choose

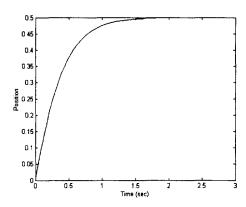


Fig. 5(a) Position trajectory of q_1

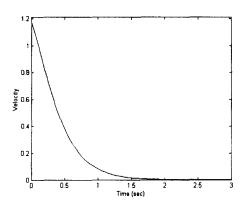


Fig. 5(b) Velocity trajectory of q_1

$$u_1 = u_2 = u \tag{42}$$

then (41) becomes

$$\begin{bmatrix} \ddot{q}_1 \\ \ddot{q}_2 \end{bmatrix} = \frac{1}{2} \begin{bmatrix} q_2^2 & -\dot{q}_1 \dot{q}_2 \\ -\dot{q}_1 \dot{q}_2 & \dot{q}_1^2 \end{bmatrix} \begin{bmatrix} u \\ u \end{bmatrix}. \tag{43}$$

Choose the output to be q_1 and \dot{q}_1 . The governing dynamics are

$$\begin{bmatrix} \ddot{q}_1 \\ \ddot{q}_2 \end{bmatrix} = \begin{bmatrix} \dot{q}_1 \\ \frac{1}{2} (\dot{q}_2^2 - \dot{q}_1 \dot{q}_2) u \end{bmatrix}$$

$$\tag{44}$$

To drive $q_1 \rightarrow q_{1d}$ and $\dot{q}_1 \rightarrow \dot{q}_{1d}$ the control law is chosen to be:

$$u = \frac{2}{\dot{q}_2^2 - \dot{q}_1 \dot{q}_2} (-k_1 (q_1 - q_{1d}) - k_2 (\dot{q}_1 - \dot{q}_{1d}))$$
 (45)

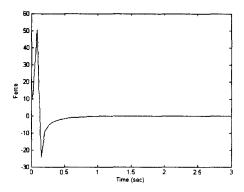


Fig. 6 Control profile of u

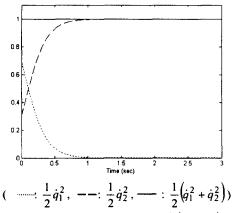


Fig. 7. Evaluation of constraint $\frac{1}{2}(\dot{q}_1^2 + \dot{q}_2^2)$

where $k_1 > 0$ and $k_2 > 0$ and q_{1d} and \dot{q}_{1d} are constants.

A simulation is performed with initial conditions as $q_1(0)=0, \dot{q}_1(0)=1.1832, q_2(0)=0,$ and $\dot{q}_2(0)=0.7746.$ It means $\dot{q}_1(0)^2=1.4$ and $\dot{q}_2(0)^2=0.6$ in order to satisfy the constraint $\frac{1}{2}(\dot{q}_1^2+\dot{q}_2^2)=1$. The desired outputs are $q_{1d}=0.5$ and $\dot{q}_{1d}=0$. Control parameters are chosen to be $k_1=16$ and $k_2=8$. Figure 5 shows that the control task is achieved satisfactorily.

Figure 6 shows the control profile of u.

Figure 7 reveals that the constraint which is nonlinear in \dot{q}_1 and \dot{q}_2 is satisfied during the motion control.

5. Concluding Remarks

It is investigated that the new modeling technique

using the Udwadia-Kalaba approach has many advantages for the motion control of constrained mechanical systems. Control of mechanical systems with many constraints including nonholomic constraint can be modeled in a unified way and results a new equations of motion where the constraints are embedded. It applies to the constraints that can be nonlinear in q, \dot{q} , and t. So far, no other technique applies to when f is nonlinear in \dot{q} . The resulting control scheme is only position and velocity feedback. It does not need force sensing (i.e., no force feedback). All other technique requires the use of Lagrange multiplier which in turn means force measurement (when one needs to feedback the Lagrange multiplier).

Because of the explicit expression of the equation of motion, one can see explicitly how the input is affected due to the presence of constraint (ref. Eq. (14), when the input matrix is changed from I to $I - M^{1/2}C^+AM^{-1}$). This shows the effect of constraint on control. Many other work (such as McClamroch and Wang⁽¹⁾ and Su and Stepanenko⁽⁹⁾) treats the Lagrange multiplier as an external signal to the system.

Two simulated examples are investigated to show the validity of the method. It should be mentioned that any elaborated control schemes can be developed more easily to achieve more complex control tasks from the proposed modeling technique.

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